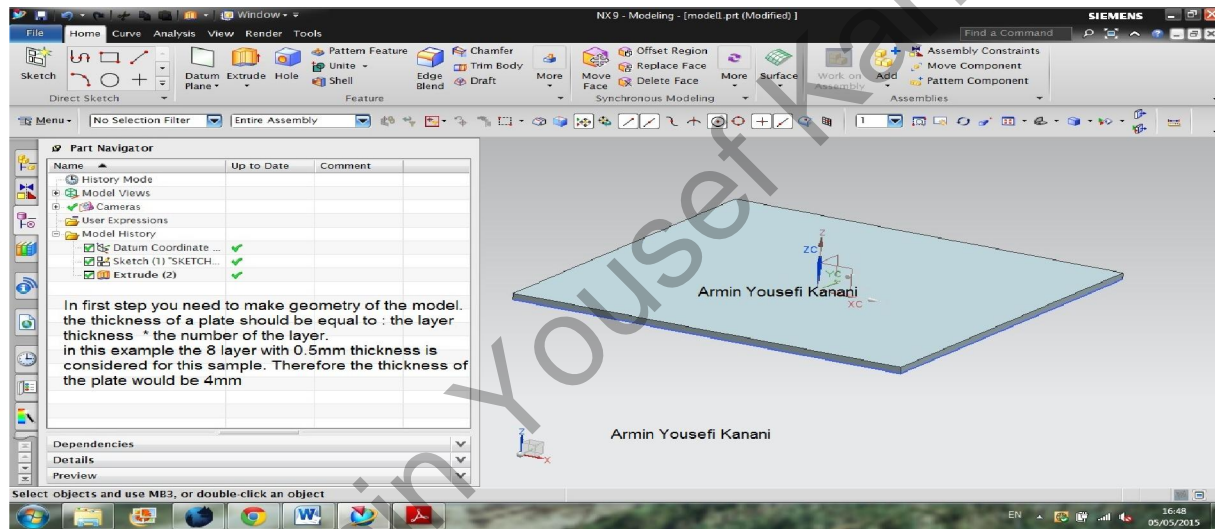


This is tutorial for buckling analysis in NX 9 software. For doing this tutorial you need to have basic information of software and also you need to be familiar with buckling subject.

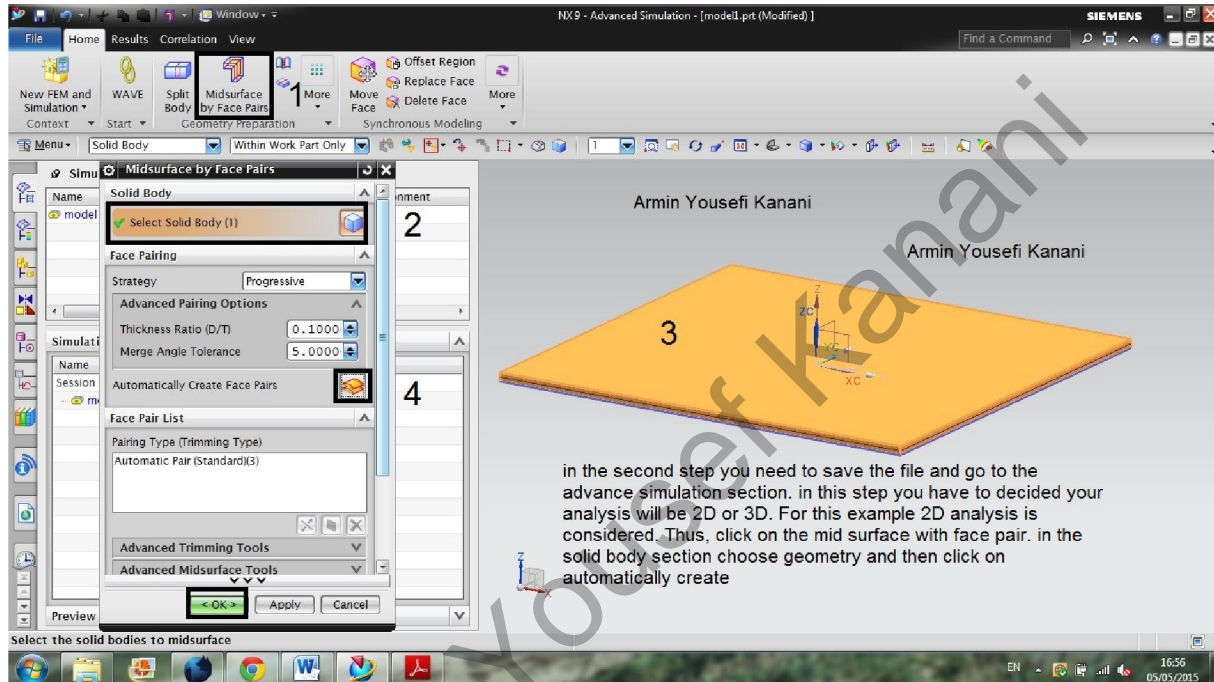
This tutorial is made by Armin Yousefi Kanani,

In the first step you need to make geometry of the model. The thickness of a plate should be equal to: the layer thickness \* the number of the layer.

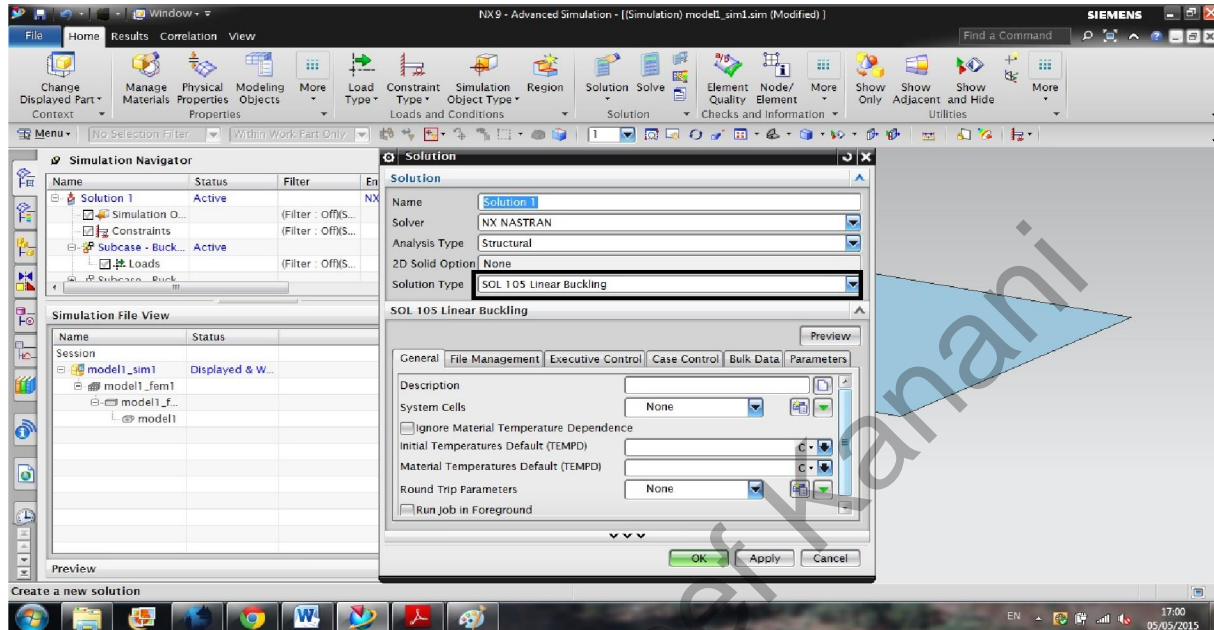
In this example the 8 layer with 0.5mm thickness is considered for this sample. Therefore the thickness of the plate would be 4mm.



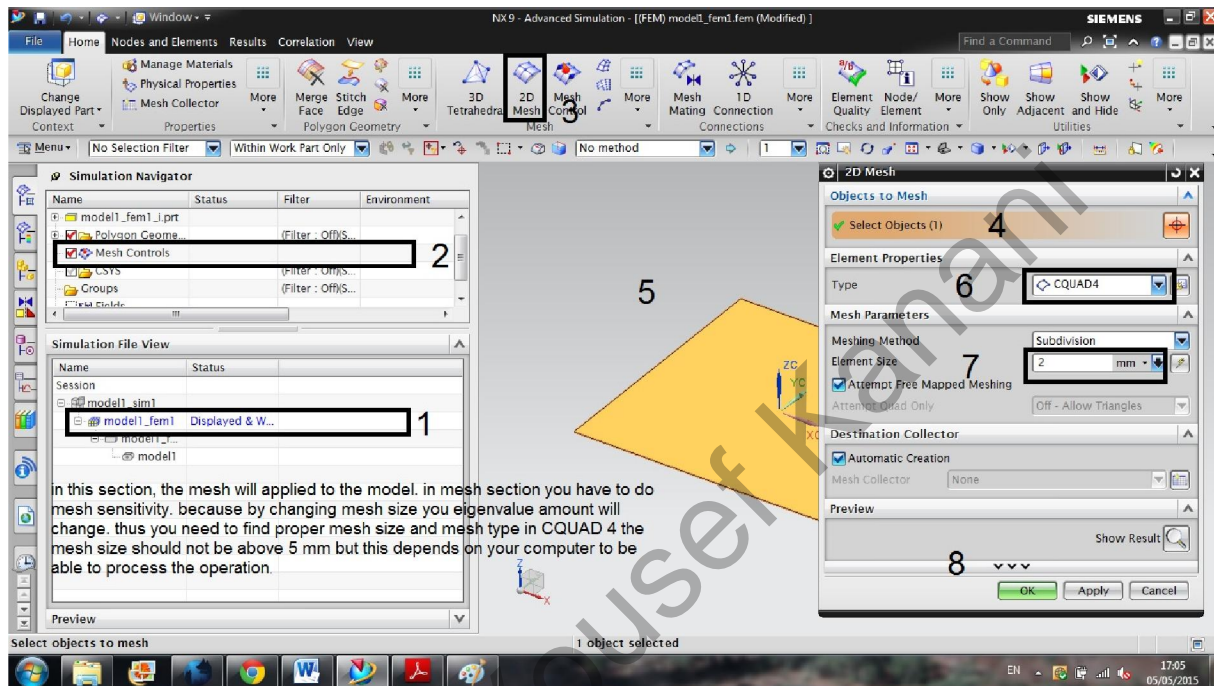
In the second step you need to save the file and go to the advance simulation section. In this step you have to decide your analysis will be 2D or 3D. For this example 2D analysis is considered. Thus, click on the mid surface with face pair. In the solid body section choose geometry and then click on automatically create



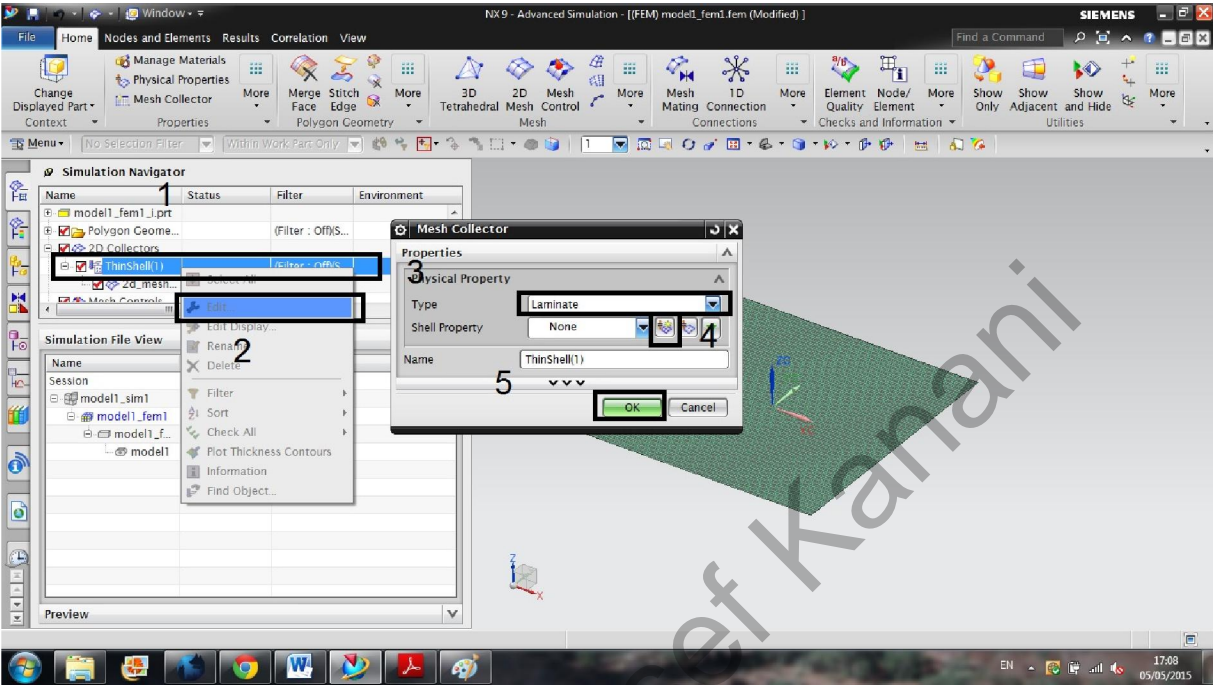
In step three you need make new simulation, for this example you do not need change any option just in the solution type choose liner buckling 105.



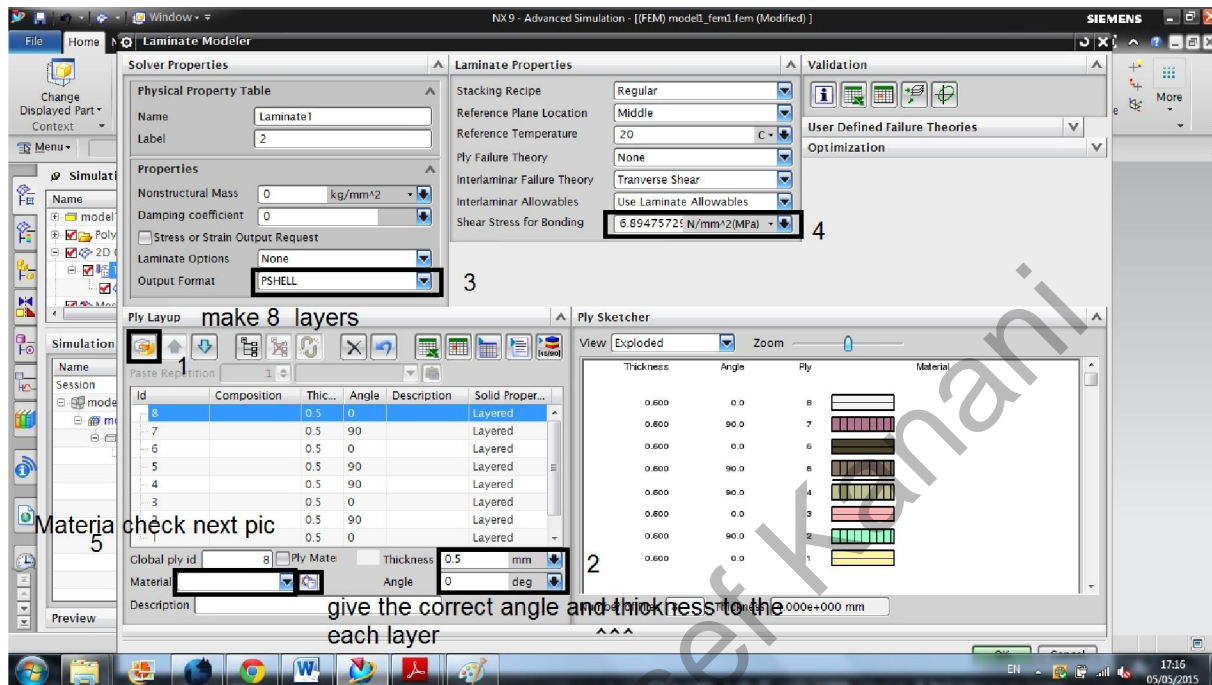
In this section, the mesh will apply to the model. In mesh section you have to do mesh sensitivity. By changing mesh size eigenvalue amount will change. Thus you need to find proper mesh size and mesh type. In CQUAD 4, the mesh size should not be above 5 mm but this depends on your computer to be able to process the operation.



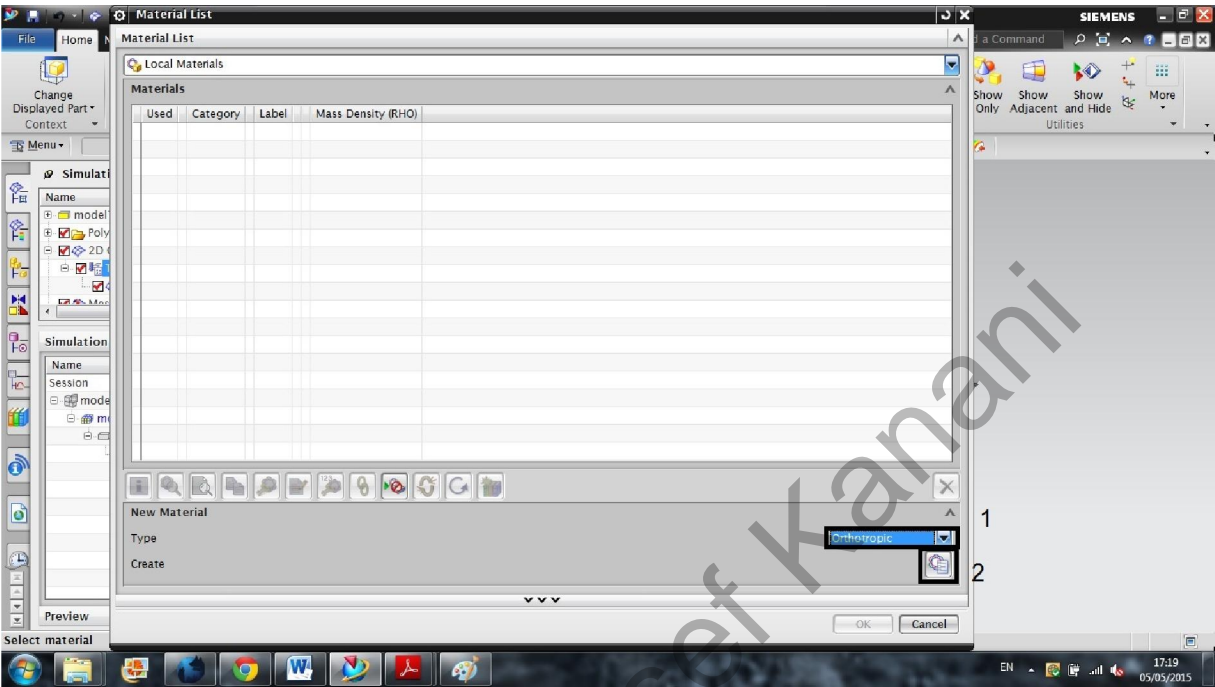
In this section we want to enter laminate information to the software.



The laminate code for this example is  $[(0, 90)_4]_s$ , so we have 8 layers with 0.5mm thickness



Material





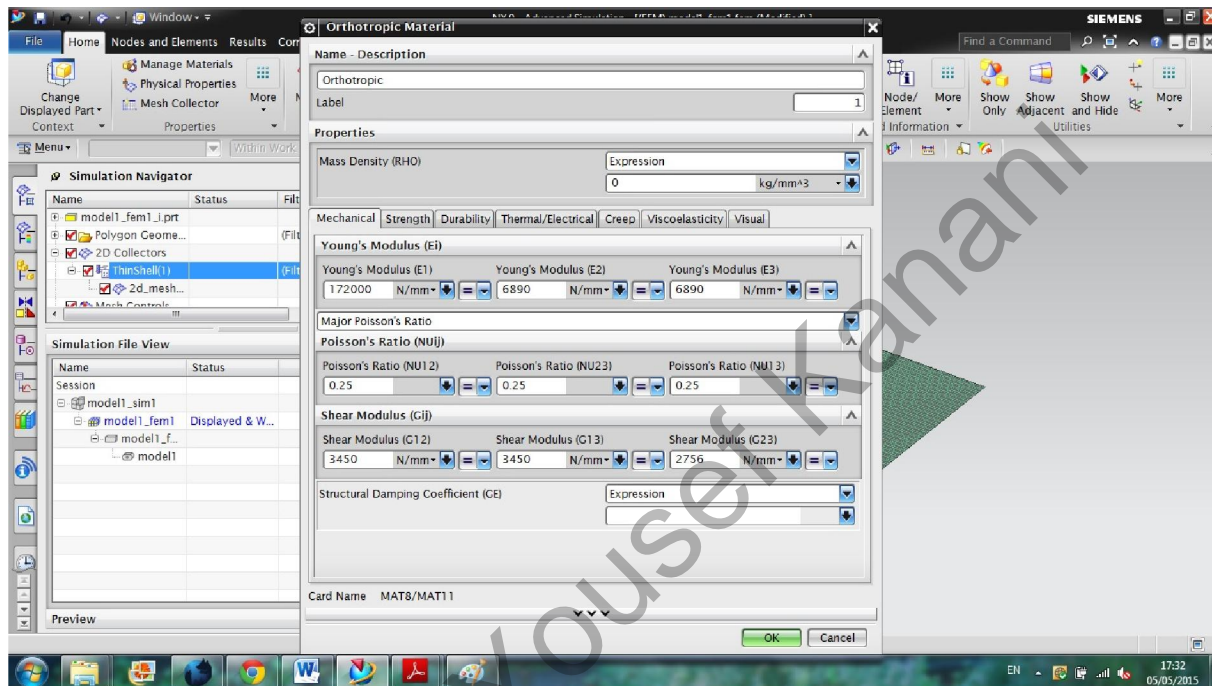
Material details

E1:172 Gpa

E2:6.89 Gpa

$\nu_{12}=\nu_{21}=0.25$

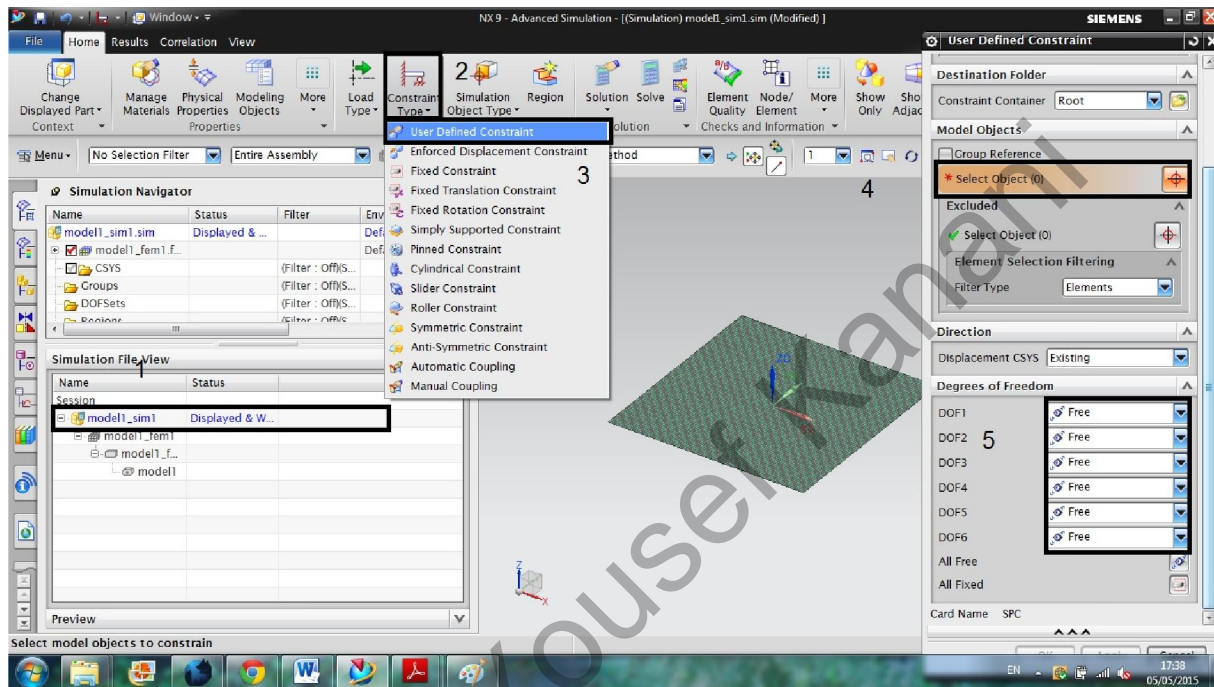
After entering information you need to give orthopedic material to each layer like previous section





In this section you need to decide about your boundary condition. The buckling test is considerably sensitive to the boundary condition. Therefore it is important to choose correct one. For gaining sufficient information I suggest you to study the R.Jones book.

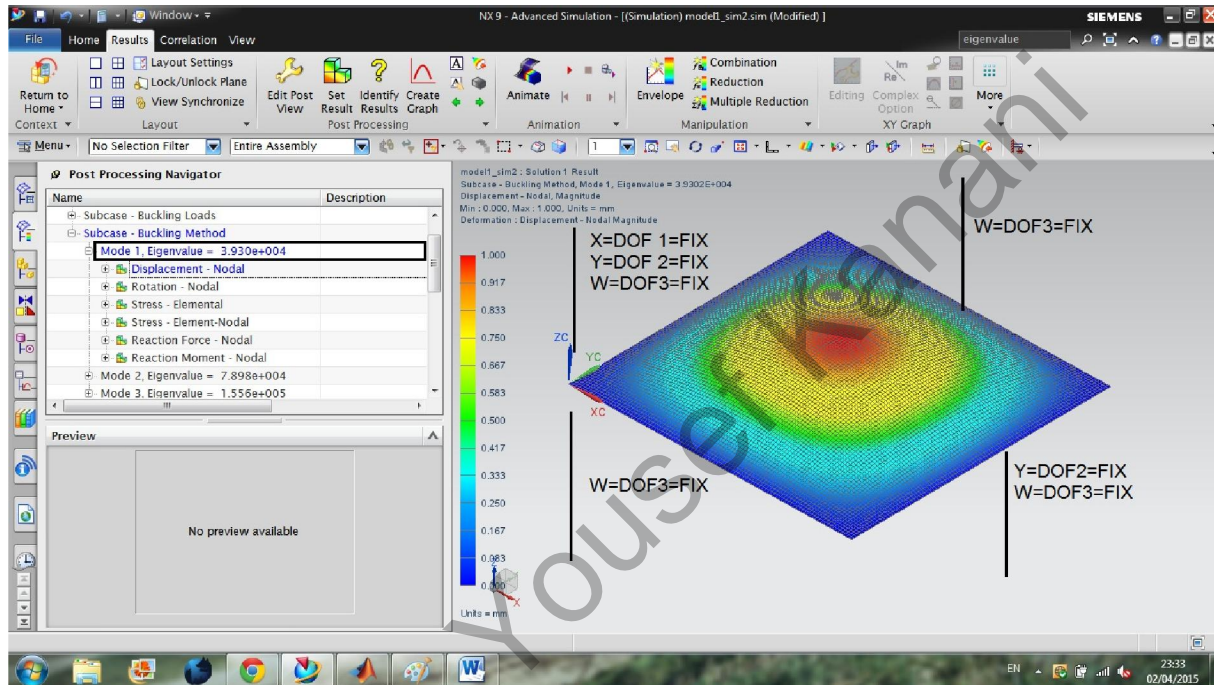
For this example simply support boundary condition will be considered



DOF1: X / DOF 2 : Y/ DOF3: Z / DOF 4:Mx / DOF 5: My/ DOF 6: Mz

For simply supported boundary condition same as the R.Jones Book. This part is significantly important, so if you want to do this tutorial please get enough information about the kind of the boundary condition.

Point: The edge should not be fixing in the applied forces direction



The screenshot displays the ANSYS Workbench interface for a buckling analysis. The main window shows a 3D visualization of the buckling mode shape for a rectangular plate. The color scale represents the magnitude of the displacement, ranging from -1.424E-003 (blue) to 1.435E-003 (red). The plot is labeled with boundary conditions:  $M_x = DOF4 = \text{FIX}$ ,  $Y = DOF2 = \text{FIX}$ ,  $W = DOF3 = \text{FIX}$ , and  $My = DOF5 = \text{FIX}$ . The 'Post Processing Navigator' on the left shows the hierarchy: Subcase - Buckling Method, Mode 1, Eigenvalue = 7.106E+004, Displacement - Nodal, X, Y, Z, Magnitude. The 'Preview' section at the bottom indicates 'No preview available'.



The first mode is critical buckling load. So, we consider first eigenvalue as the critical buckling load

